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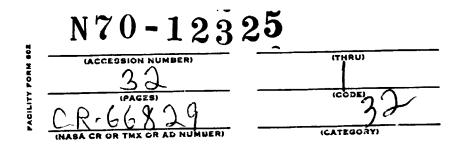
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# SOME ASPECTS OF FATIGUE CRACK PROPAGATION

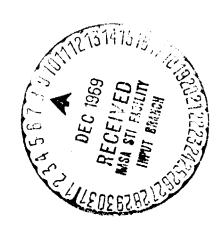
by

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#### SOME ASPECTS OF

FATIGUE CRACK PROPAGATION

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#### SUMMARY

The models of fatigue crack propagation proposed by Forman et al and Roberts and Erdogan were studied in this paper. By applying these models to existing data in the literature for thin 2024-T3 and 7075-T6 aluminum rlates subjected to fluctuating tensile loads, it was found that both models gave comparable results when one considered just a gross correlation of the experimental By modifying Forman's model to incorporate the ideas of Roberts and Erdogan, a model was produced which appeared to be a more rational basis for studying the problem of fatigue crack propagation in thin plates and shells subjected to tensile loads, bending loads, or a combination of both. This fact was demonstrated for the case of thin plates subjected to fluctuating bending loads and for the case of thin cylindrical shells subjected to fluctuating internal pressure.

This paper also presents the large quantity of data relating the rate of fatigue crack propagation in thin plates subjected to fluctuating bending loads collected at Lehigh University.

#### INTRODUCTION

Recent work (refs. 1, 2, and 3) has clearly shown that the rate of fatigue crack propagation in thin sheets subjected to either fluctuating tensile loads or fluctuating bending loads is related to the stress-intensity factor.

It is the purpose of this paper to evaluate the fatigue crack propagation models of Forman et al (ref. 2) and Roberts and Erdogan (ref. 3). This evaluation will be limited to two aluminum alloys, 2024-T3 and 7075-T6. The primary source of data for fatigue crack propagation due to plane extension used in this evaluation will be the work of Donaldson and Anderson (ref. 4) and the work of Schijve and his co-workers (ref. 5). The data relating the effect of bending loads on fatigue crack propagation comes from the extensive work performed at Lehigh University during the past few years.

Since the large quantity of data gathered at Lehigh University relating the rate of fatigue crack propagation to fluctuating bending loads is not readily available in the literature, it is the secondary purpose of this paper to present these data in a convenient form whereby other researchers may make use of these data. To date, none of

the raw data and only a partial analysis of some of the data appears in the literature (refs. 3 and 6).

#### FATIGUE CRACK PROPAGATION MODELS

Many functional forms have been proposed to represent the relationship between the rate of fatigue crack propagation and various physical parameters.

Paris and Erdogan (ref. 1) proposed the following relationship between the rate of fatigue crack propagation and the stress-intensity factor amplitude,  $\Delta K$ :

$$\frac{da}{dN} = A(\Delta K)^{m} \tag{1}$$

where a is the half crack length for a center-cracked specimen, N is the number of load cycles,  $\Delta K$  is the stress-intensity factor amplitude defined herein as  $\Delta K = (K_{\text{max}} - K_{\text{min}})/2$ , A is a constant which must be determined for each material, mean load, etc., and m is a numerical exponent. Paris and Erdogan found the general trend of the data indicated that m  $\cong$  4.

Forman et al (ref. 2) argued that a correct crack growth law should include the criterion that the crack growth rate approach infinity as the maximum value of the stress-intensity factor,  $K_{\rm max}$ , approaches the stress-intensity

level for rapid fracture,  $K_{\text{c}}$ . Thus by modifying the model of Paris and Erdogan, equation 1, Forman et al proposed an equation of the form

$$\frac{da}{dN} = \frac{B(\Delta K)^{n}}{(1-R)K_{C}-2\Delta K}$$
 (2)

where  $R = K_{min}/K_{max}$ ,  $K_{C}$  is the critical stress-intensity factor for rapid fracture, B is a material constant, and n is a numerical exponent.

In evaluating their model, Forman et al used the data found in (refs. 4, 5, and 7). For the aluminum alloys 2024-T3 and 7075-T6, they found n = 3. They also found the values of  $K_C$  which best fit the data were in reasonable agreement with published values of  $K_C$ .

Erdogan and Roberts (refs. 3 and 6) proposed that the rate of fatigue crack propagation is probably more fundamentally related to the size of the plastic zone ahead of and in the plane of the fatigue crack. This led to an equation of the form

$$\frac{da}{dN} = C(K_{max})^{p}(\Delta K)^{q}$$
 (3)

where C is a material constant, and p and q are numerical exponents. They found for tensile and bending loads that the values of p and q in equation (3), for 2024-T3 and

7075-T6 aluminum alloys, could be approximated as  $p = q \approx 2$ .

#### EXPERIMENTAL RESULTS

The results of two experimental programs are reported in this paper. In the first program, 2024-T3 and 7075-T6 aluminum plates which contained a central through crack were subjected to fluctuating cylindrical bending loads. Reference (6) gives a detailed description of the testing equipment and procedures used for collecting the data. Appendix A contains the values of the crack length, number of cycles, specimen thickness, load levels, etc., for these tests.

The second program consisted of subjecting 2024-T3 bare aluminum plates containing a central through crack to a combined static tensile load and a cyclic transverse bending load. Reference (12) gives a detailed description of the testing equipment and procedures. The results obtained from the combined loading tests are given in Appendix B.

The mechanical properties of the combined loading specimens and the cylindrical bending specimens are given in Appendix C. The values listed in this appendix are the

average values for four test specimens. These mechanical properties were determined in accordance with the appropriate ASTM standard for sheet material.

#### DISCUSSION

The bending data given in the Appendices will not be discussed in detail in this paper. These data will be used solely as a means of comparing the models of Forman et al and Roberts and Erdogan.

The quantity  $\beta$  was introduced into equations (2) and (3) before making the comparison of these equations.  $\beta$  is defined as  $\beta = K_{\text{mean}}/\Delta K$  where  $K_{\text{mean}} = (K_{\text{max}} + K_{\text{min}})/2$ . Thus equations (2) and (3) can be written as

$$\frac{\mathrm{da}}{\mathrm{dN}} = \frac{\mathrm{B}(1+\beta)(\Delta K)^{\mathrm{n}}}{\mathrm{K}_{\mathrm{C}} - (1+\beta)\Delta K} \tag{4}$$

and

$$\frac{da}{dN} = C(1 + \beta)^{p} (\Delta K)^{p+q}$$
 (5)

To compare equations (4) and (5) the logarithm of both sides of equations (4) and (5) were taken. The resulting equations, linear in B, C, n, p, and q, were fit to the available data using a least squares fit program. The program calculated the standard error,

 $S_{y,x}$  , along with the values of B, C, n, p, and q. For this program

$$S_{y,x} = \sqrt{\frac{\sum_{i=1}^{M} \left[ \ln \left( \frac{da}{dN} \right)_{i \text{ observed}} - \ln \left( \frac{da}{dN} \right)_{i \text{ calculated}} \right]^{2}}$$
M
(6)

where M is the number of data pairs used in the curve fit. It was decided that the standard error was the best means of comparing the two equations. The equation which consistently had the smaller standard error would be the one which best fit the data. Since the standard error is being used as the measure of comparison, figures showing how equations (4) and (5) fit the experimental data were not prepared. The reader is referred to references (2), (3), and (6) for such figures.

Unfortunately or fortunately, depending on one's point of view, both equations gave almost identical results. The standard errors associated with fitting the values of B and C to equations (4) and (5) for the values of K<sub>C</sub> and n recommended by Forman et al and the values of p and q recommended by Roberts and Erdogan are given in Tables 1 and 2. By comparing the values of S in these y,x tables, it is seen that the two equations give comparable

results. It should be noted that the best fit values of B, C, n, p, and q, along with the associated standard error were calculated for equations (4) and (5). These results, although not included in this paper, indicate that the value of n=3 recommended by Forman et al, and p=q=2 recommended by Roberts and Erdogan were in reasonable agreement with the least squares fit of the data. The standard errors for this case again showed that the two equations give comparable results.

It is the opinion of the present authors that the assumption made by Forman et al is correct in view of the data considered to date. The reasons for this are as follows:

- The equation proposed gives excellent results when predicting the effect of mean stress on fatigue crack propagation due to plane extension for the two aluminum alloys considered.
- 2) The value of  $K_C$  in the equation of Forman et al is in good agreement with values of  $K_C$  obtained from fracture tests.

It is also the opinion of the present authors that the assumptions put forward by Roberts and Erdogan are correct. One reason for this is the ability of their model to handle the effect of mean stress on fatigue crack propagation. A second and more important reason is the ability of their model to predict the rate of fatigue crack propagation due to bending loads from data obtained in tensile tests.

By using the argument that fatigue crack propagation is more fundamentally related to plastic zone size and by observing the similarity of fracture modes between the cylindrical bending tests given in Appendix A and results from plane extension, Roberts and Erdogan argued that for the same material if the plastic zone sizes for the two types of loading are the same, the growth rates should be the same. This led to the conclusion that the rate of fatigue crack propagation due to bending loads could be predicted from data obtained from tension testing. By modifying the stress-intensity factor for bending by a factor of 1/2 and using the value of C in equation (5) obtained from tensile data, one has

$$\frac{\mathrm{da}}{\mathrm{dN}} = C(1 + \beta)^2 \left(\frac{\Delta K_b}{2}\right)^4 \tag{7}$$

where  $\Delta K_b$  is the amplitude of the bending stress-intensity factor. The factor 1/2 was determined theoretically so that the plastic zone sizes for both cases would be the same. Equation (7) was considered in reference (3).

Excellent results were obtained in predicting the fatigue crack propagation rates associated with the data given in Appendix A from fatigue crack propagation rates found in the literature for tensile data.

In a recent study, Catanach (ref. 10) investigated the rate of fatigue crack propagation in thin 6063-T6 aluminum shells. In his tests he subjected the shells which contained a longitudinal through crack to fluctuating internal pressure. As a result of the shell curvature the area near the crack tip is simultaneously subjected to varying tensile and bending loads. Using the results of Roberts and Erdogan (refs. 3 and 6), it can be shown that the plastic zone size can be approximated in the Dugdale sense (ref. 11) as being proportional to the square of  $K_t + K_b/2$ , where  $K_t$  is the stress-intensity factor for the in-plane tensile loads and  $K_{\dot{b}}$  is the stress-intensity factor for the bending loads. Catanach fit equation (5) to his data. He found that he got good results using a value of  $\Delta K = \Delta K_t + \Delta K_b/2$  in equation (5). Unfortunately the levels of  $v_b$  compared to  $K_t$  were so small that it was impossible to tell if the quantity  $K_t + K_b/2$  was the reason for the good fit.

Based on the previous thoughts concerning the models

of Forman et al and Roberts and Erdogan and the work of Catanach, an equation of the form

$$\frac{da}{dN} = D\left(\frac{K_{max}}{K_{c} - K_{max}}\right)^{r} (\Delta K)^{s}$$
 (8)

was fit to the data used to compare equations (4) and (5). The best fit values of r and s were r = 1, s = 3. Thus fixing the values of r and s at 1 and 3 respectively, one arrives back at Forman's equation:

$$\frac{da}{dN} = \frac{B(1 + \beta)(\Delta K)^3}{K_C - (1 + \beta)\Delta K}$$

The hypothesis of Roberts and Erdogan about the relationship between fatigue crack propagation rates due to tension and bending was tested by writing equation (4) as

$$\frac{da}{dN} = \frac{B(1 + \beta) (\frac{\Delta K_b}{2})^3}{K_c - (1 + \beta) (\frac{\Delta K_b}{2})}$$
(9)

This equation was fit to the data found in Appendix A for the 0.05 inch thick 2024-T3 aluminum for  $\beta$  = 0.392, 0.632, and 1.0. The value of B found was 2.78 x  $10^{-12}$ . This compares very well with the value of B given in Table 3 for the 0.04 inch material from reference (5),  $C = 2.94 \times 10^{-12}$ .

As a result of the above considerations, the following modification of Forman's equation is proposed:

$$\frac{\mathrm{da}}{\mathrm{dN}} = \frac{\mathrm{C}(1+\beta)(\Delta K_{\mathrm{e}})^{3}}{\mathrm{K}_{\mathrm{c}} - (1+\beta)\Delta K_{\mathrm{e}}}$$
(10)

where Ke is defined as

 $K_e = K_t$ ; for plane extension

 $K_e = K_b/2$ ; for bending

 $K_e = K_t + K_b/2$ ; for combined loading

and  $K_{\rm t}$  and  $K_{\rm b}$  are the stress-intensity factors for extension and bending, respectively. This equation incorporates the concept of Forman et al that the fatigue crack propagation rate should become infinite as  $K_{\rm max}$  approaches  $K_{\rm c}$ . It also incorporates the concept of Roberts and Erdogan that for similar fracture modes but different types of loading, the rate of fatigue crack propagation should be the same if the plastic zone sizes are the same.

With regard to the transverse bending tests, a suitable model for estimating the plastic zone size que to the combined axial and transverse load has not been developed. When this is done a method for determining  $K_e$  in equation (10) will be available.

#### SUMMARY OF RESULTS

- 1. The equations proposed by Forman et al and Roberts and Erdogan both handle the effect of mean stress on fatigue crack propagation equally well.
- 2. The following equation, a modification of Forman's equation, is proposed in place of the equations of Forman et al and Roberts and Erdogan:

$$\frac{da}{dN} = \frac{B(1 + \beta)(\Delta K_e)^3}{K_c - (1 + \beta)\Delta K_e}$$

where  $K_e$  is defined as

 $K_{e} = K_{i}$ ; for plane extension

 $K_e = K_b/2$ ; for bending

 $K_e = K_t + K_b/2$ ; for combined loading

and  $K_t$  and  $K_b$  are the stress-intensity factors for extension and bending, respectively.

In closing, the authors would like to point out that the equation proposed was evaluated for only two aluminum alloys. Other materials might not show agreement with the equation. The hypothesis that  $K_e = K_t + K_b/2$ 

must be viewed with caution until it can be compared to more substantial data than found in reference (10). In general a large number of questions are not answered by equation (10) and should be the object of future studies.

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TABLE 1

Material Constant and Standard Error for the Equation of Roberts and Erdogan

$$\frac{da}{dN} = C(1 + \beta)^2 \Delta K^4$$

Material	Reference	Thicknes <b>s</b>	$C \times 10^{-21}$	s <sub>y,x</sub>
2024 <b>-</b> T3	5	0.080 0.024 0.040 0.079 0.118 0.157	15.89 7.48 7.49 6.52 10.26 12.51	0.6167 0.7444 0.4112 0.5680 0.3144 0.3802
	4	0.020 0.032 0.040 0.063 0.081 0.102	6.92 8.12 5.73 7.49 4.26 10.19	0.6153 0.3266 0.4017 0.4807 0.4892 0.6469
	7	0.081	5.01	0.5503
	5	0.080	82.27	0.4189
7075 <b>-</b> T6	4	0.064 0.090 0.102	22.20 23.81 24.41	0.4150 0.1717 0.5888
	7	0.081	5.31	0.5527
	9	0.090	11.23	0.5818

TABLE 2

Material Constant and Standard Error for the Equation of Forman et al.

$$\frac{da}{dN} = \frac{B(1 + \beta)\Delta K^3}{K_C - (1 + \beta)\Delta K}$$

2024-T3 ;  $K_c = 47,000$ 7075-T6 ;  $K_c = 38,500$ 

Material	Reference	Thickness	C x 10 <sup>-12</sup>	s <sub>y,x</sub>
2024 772	5	0.080 0.024 0.040 0.079 0.118 0.157	2.69 2.89 2.94 2.60 4.09 5.02	0.3670 0.7314 0.4355 0.4546 0.2809 0.4006
2024-T3	4	0.020 0.032 0.040 0.063 0.081 0.102	2.55 3.16 2.20 2.08 1.07 3.30	0.5310 0.4223 0.2652 0.2730 0.2517 0.5109
	7	0.081	1.53	0.5601
	5	0.080	10.07	0.3489
70 <b>75-</b> T6	4	0.064 0.090 0.102	6.43 7.06 5.28	0.3327 0.2835 0.3278
	7	0.081	1.22	0.3257
	9	0.090	3.02	0.5466

Crack Growth Data For Cylindrical Bending

APPENDIX A

Plate Number	Material	Thickness (in.)	O-a (PSI)	$\beta = (O_m/O_a)$
1	2024-T3 Clad	0.050	12,500	0
2	2024-T3 Clad	0.050	16,640	Ö
	2024-T3 Clad	0.050	18,850	0
3 4	2024-T3 Clad	0.050	18,850	Ö
	2024-T3 Clad	0.050	18,850	Ŏ
6	2024-T3 Bare	0.050	12,175	0.39
5 6 7	2024-T3 Bare	0.050	12.175	0.39
8	2024-T3 Bare	0.050	12,435	0.62
9	2024-T3 Bare	0.050	12,435	0.62
10	2024-T3 Bare	0.050	8,320	1
11	2024-T3 Bare	0.050	8,320	ī
12	2024-T3 Bare	0.080	21,800	ō
13	2024- <b>T</b> 3 Bare	0.080	26,000	Ŏ
14	2024-T3 Bare	0.080	30,200	Ō
15	2024-T3 Bare	0.080	30,200	Ö
16	2024-T3 Bare	0.080	30,200	Ō
17	2024-T3 Bare	0.080	30,200	Ō
18	2024-T3 Clad	0.080	10,400	0
19	2024-T3 Bare	0.100	26,000	0
20	2024-T3 Bare	0.100	26,000	0
21	2024-T3 Bare	0.100	20,000	0
22	2024-T3 Bare	0.100	30,700	0
23	2024-T3 Bare	0.100	35,400	0
24	2024-T3 Bare	0.100	35,400	0
25	2024-T3 Clad	0.100	13,000	0
26	2024-T3 Clad	0.100	35,000	0
27	2024-T3 Clad	0.100	38,500	0
28	2024-13 Bare	0.125	16,100	0
29	2024-T3 Bare	0.125	18,400	0
30	2024-T3 Bare	0.125	18,400	0
31	2024-T3 Clad	0.125	31,200	0
32	2024-T3 Clad	0.125	37.400	0
33	2024-T3 Clad	0.125	44,700	0
34	2024-T3 Clad	0.160	19 <b>,75</b> 0	0
35	2024-T3 Clad	0.160	19,750	0
36	2024-T3 Clad	0.160	19,750	0
37	2024-T3 Clad	0.160	19,750	0
38	2024-T3 Clad	0.160	23, 100	0
39	2024-T3 Clad	0.160	29,600	0
40	7075-T6 Clad	0.050	21,800	0
41	7075-T6 Clad	0.050	21,800	0
42	7075-T6 Clad	0.050	29, 800	0
43	7075-T6 Clad	0-100	28,000	0
44	7075-16 Clad	0.100	28,000	0
45	7075-16 Clad	0-100	28,000	0
46	7075-16 Clad	0.100	34 <b>, 4</b> 00	0
47	7075-16 Clad	0.100	34 <b>, 4</b> 00	0
48	7075-T6 Clad	0.100	44, 200	0
49	7075-T6 Clad	0.100	44,200	0

# APPENDIX A (cont.)

Plate	Material	Thickness	(DOT)	$\beta = \langle \mathcal{O}_{\mathbf{m}} / \mathcal{O}_{\mathbf{n}} \rangle$
Number		(in.)	(PSI)	
50	7075-T6 Clad	0.100	52,500	0
51	7075-T6 Bare	0.120	14,80C	0
52	7075-T6 Bare	0.120	31,200	0
53	7075-T6 Bare	0.120	34,800	0
54	7075-T6 Bare	0.120	42,500	0

Plate #1	Plate #3 cont.	Plate #5 cont.	Plate #7
2a Nx10 <sup>3</sup>	2a Nx10 <sup>3</sup>	2a Nx10	2a Nx10 <sup>3</sup>
0.687 325	0.553 91	1.107 137	0.255 0
0.704 340	0.600 100	1.247 140	0.260 6
0.713 365	0• <i>65</i> 9 110	1.319 142	0.268 12
0.755 395	0.711 120	1.330 143	0.313 36
0.814 419	0.854 130	1.431 145	0.348 59
0.878 430	1.047 135	1.589 147	0.382 77
0.900 445	1.107 137	1.664 148	0.384 95
0.974 460	1.247 140		0.402 118
1.007 475	1.319 142		0.420 140
1.069 490	1.330 143	Plate #6	0.439 160
1.083 505	1.431 145	3	0.452 178
1.154 520	1.589 147	2a Nx10 <sup>3</sup>	0.478 200
1.226 535	1.664 148	0.238 0	0.511 224
1.244 550		0.241 6	0.542 243
1.298 565	· <b>#</b>	0.243 12	0.579 262
1.570 580	Plate #4	0.256 36	0.619 282
1.641 585		0.278 59	0.649 298
	2a 1x10 <sup>3</sup>	0.281 77	0.695 316
#-	0.271 25	0.287 95	0.748 332
Plate #2	0.288 35	0.304 116	0.807 347
3	0.302 45	0.319 140	0.899 364
2a 1210	0-368 60	0.336 160	0.952 374
0.328 55	0.413 70	<b>0.35</b> 0 178	1.060 389
0.353 65	0.451 80	0.377 200	1.195 405
0.407 75	0.485 90	0.403 224	1.328 417
0.423 85	0.541 100	0.419 243	1.497 429
0.460 95	0.594 110	0.442 262	1.552 433
0.510 105	0.710 120	0.468 282	,,
0.549 117	<b>0.937</b> 130	0.489 298	
0.597 130	1.053 133	0• <i>5</i> 16 316	Plate #8
0.638 145	1.096 135	0.546 332	وتسليستيد تدريق
0.793 160	1.120 140	0•576 347	2a Mx10 <sup>3</sup>
0.855 170	:476 145	0.619 3 <i>6</i> 4	0.282 155
0.883 180	1. <i>56</i> 9 '45	0.646 374	0.296 175
0.984 190	1.768 149	0• <i>6</i> 93 389	0.312 190
1.067 200		0.748 405	0.340 205
1.141 210	_	0.800 417	0.368 21.8
1.225 215	Plate #5	0.860 429	0.393 230
1.465 220	3	0.941 443	0.419 240
1.563 225	2a Hx10	1.095 461	0.449 250
1.782 230	0.361 50	1.195 471	0.478 260
	0.401 60	1.296 479	0-505 270
	0.486 70	1.393 486	0.538 280
Plate #3	0.502 80	1.454 490	0.578 290
	0.553 91	1.513 494	0.630 300
2a 110 <sup>3</sup>	0.600 100	1.592 498	0.683 310
0.361 50	0.659 110		0.748 320
0.401 60	0.711 120		0.827 330
0.486 70	0.854 130		0.922 340
0.502 80	1.047 135		1.039 350
			-

Plate #8 cont.	Plate #11	Plate #13 cont.	Plate #16 cont
2a Nx103	30 15-103	3- 7-103	3
1.182 360	2a <u>Nx10<sup>3</sup></u> 0.232 0	2a Nx10 <sup>-</sup>	2a Nx10 <sup>3</sup>
1.329 369	0.245 90	1.032 25 1.432 28	1.47) 19.5
1.460 374	0.267 200	1.432 28 1.746 29	1.670 20.0
1.704 384	0.354 342	1.740 29	
	0.432 514		Plate #17
	0.518 600	Plate #14	TIGAG ATL
Plate #9	0.546 650	11000	2a Nx10 <sup>3</sup>
2	0.642 730	$2a Nx10^3$	0.280 1
2a Nx10	0.711 790	0.299 1	0.319 2.25
0.303 90	0.741 810		0.341 3
0.363 125	0.835 835	0.366 3 0.432 5 0.509 7	0.366 4
0.436 155	0.886 860	0.509 7	0.434 6
0-495 175	0.889 880	0.604 9	0.528 8
0.549 190	<b>0.</b> 936 895	0.720 11	0.658 10
0.619 205	1.011 915	0.904 13	1.048 13
0.697 218	1.054 927	1.317 15	1.428 14
0.784 230	1.121 941	1.580 15.8	1.674 14.3
0.873 240	1.166 955	1.744 16.25	
0.921 245	1.265 970		
0.978 250	1.362 986	_	Plate #18
1.031 255	1.476 1000	Plate #15	2
1.099 260	1.566 1020	3	2a Nx10 <sup>3</sup>
1•174 265 1•222 268		2a Nx10 <sup>3</sup>	0 <b>. 27</b> 1 700
	Mata Iso	0.231 0	0.294 850
1•254 270 1•301 272	Plate #12	0.261 1	0.302 878
1.348 275	2a Nx10 <sup>3</sup>	0.287 2	0.310 900
1.458 280	$\frac{2a}{0.299} \frac{Nx10^2}{5}$	0.346 4	0.320 927
1.538 284	0.326 10	0•385 6 0•441 8	0.330 953
1.632 287	0.356 15		0.341 986
1.716 290	0.424 25	0•535 11 0•689 15	0•347 1,000 0•367 1,049
	0.518 35	0.794 17	0.395 1,113
	0.648 45	0.927 19	0.438 1,186
Plate #10	0.841 55	1.096 21	0.452 1,215
<del></del>	1.155 65	1.223 22	0.489 1,269
2a Nx10 <sup>3</sup>	1.867 76	1.302 22.5	0.519 1,317
0.242 200		1.527 23.5	0.570 1,394
0.358 514			0.668 1,497
0.419 600	Plate #13		0.765 1,575
0.566 650	3	Plate #16	0.836 1,631
0.600 780	2a Nx10 <sup>3</sup>		0.879 1,661
0.694 790	0.238 0	2a Nx10 <sup>3</sup>	0.978 1,731
0.819 810	0.279 1	0.281 1.0	1.081 1,800
0.886 835	0.318 3	0.310 2.5	1.222 1,869
0.951 860	0.279 1 0.318 3 0.350 5 0.382 7	0.364 5.0	1.323 1.910
1.048 880	-	0.423 7.5	1.444 1.950
1.134 895	0.436 10	0.516 11.0	1.480 1,960
1.232 915	0.504 13	0.598 13.0	1.523 1.971
1.421 927	0.581 16	0.787 16.0	1.560 1,980
1.532 941	0.683 19	1.153 18.5	
1.823 955	0.823 22	1.295 19.0	

Plate	<u>#19</u>	Plate #	21	Plate	#22 cont.	Plate	#25 cont
2 <u>a</u>	Nx10 <sup>3</sup>	_2a	Nx10 <sup>3</sup>	_2a_	Nx10 <sup>3</sup>	2a	Nx10 <sup>3</sup>
0.289	2	0.256	0.5	0.763	11	0.454	600
0.346	5 6 7	0.266	1.0	0.850	12	0.530	703
0.363	6	0.278	1.5	0.944	13	0.569	750
0-382	7	0.290	2.0	1.051	14	0.621	800
0.401	8	0.310	3.0	1.171	15	0.744	905
0.422	9	0• 328	4	1.300	16	0.397	995
0.439	10	0.345	5	1.443	17	1.046	1,052
0.488	12	0.361	5 6	1.609	18	1.117	1,071
0-532	14	0•378	7	1.758	19	1.228	1,100
0.593	15	0.396	8		-,	1.336	1, 120
0.654	18	0.419	9			1.410	1,135
0.719	20	0.436	10	Plate	#23	1.477	1,146
0.792	22	0.464	11			, ,	_,
0.889	24	0.483	12	_2 <b>a</b>	Nx10 <sup>3</sup>	•	
0.982	26	0.508	13	0.249	0-50	Plate	<b>#</b> 26
1.033	27	0.532	14	0.282			120
1.096	28	0.560	15	0.344	1 2	2a	$Nx10^3$
1.161	29	0.587	16	0.403	3	0.400	2
1.235	30	0.618	17	0.727	6	0.583	Ž.
1.316	31	0.652	18	0.991	7	1.035	4 6
1.408	32	0.688	19	1.254	7.6		6
1.516	33	0.735	20	102,74	7.0	1.706	6.5
1.647	34	0.767	21				
1.798	35	0.806	22	Diata.	421	<b>D1</b> 4	# OF3
10170	7,7	0.852	23	Plate :	F 24	Plate:	<u> </u>
		0.900	24	2a	Nx103	0-	<b>-</b> 3
Plate	#20	0.950			NXIO	2a	Nx10
220,00		<b>0.</b> 999	25 26	0.322	0.50	0.318	1.0
2a	Nx103	1.059		0.370	1.00	0.401	2.0
0.850	2.0	1.121	27 28	0.410	1.50	0.459	3.0
0.928	3.0	1.121	28	0.457	2.00	0.624	4.5
1.011	4.0	1.272	29 30	0.508	2.50	0.802	5.5
1.056	4.5	1.272	<b>3</b> 0	0.562	3·00	0.978	6.2
1.105	5.0	1.478	31 32	0.637	3•50	1.067	6.4
1.158	5.5	1.593	33	0.702	4.00	1-193	6.6
1.208	6.0	1.766	34	0.796	4.50	1.343	6.8
1.264	6.5	1.100	<b>)</b> 4	0.853	4.75	1.633	7.0
1.321	7.0			0.913	5.00		
1.378	7.5	Plate #2	22	0.984	5.25		
1.442	8.0	riave T		1.082	5.50	Plate #	28
1.503	8.5	_2a1	1×10 <sup>3</sup>	1.254	5.80	•	3
1.564	9•0	0.258		1.451	6.00	<u> 2a                                   </u>	Nx10
1.621	9.5	0.297	1			0.298	82
1.682	10.0	0.334	2	maka J	100	0.323	98
1.737	10.5	0.368	2 3 4	Plate		0.338	107
<i>()</i>	1000	0.411		20	<u>N×10<sup>3</sup></u>	0.361	122
		0.522	5 6	28	TXTO	0.406	145
		0.508	7	0.203	255	0.434	164
		0• 559	8	0.297	277	0.457	169
		0• 623	9	0.303	300	0.491	188
		0.685		0.346	400	0.512	197
		U• UUJ	10	0•398	<i>5</i> 00	0.551	214

Plate	#28 cont.	Plate:	#30 cont.	Plate #	33	Plate #	
2 <b>a</b>	Nx10 <sup>3</sup>	2 <b>a</b> .	Nx10 <sup>3</sup>	28	Nx10 <sup>3</sup>	2 <b>a</b>	Nx10 <sup>3</sup>
0.587	226	0.445	88	0.297	0.1	0.468	55
0.626	240	0.500	100	0.347	042	0.494	60
0.700	261	0.600	115	0.422	0.4	0.561	68
0.767	275	0.649	121	0.495	0.6	0.616	76
0.811	285	0.749	131	0.581	0.8	0.691	84
0.863	294	0.811	136	0.729	1.0	0.745	88
0.932	306	0.883	141	0.856	1.1	0.784	92
1.004	316	0.988	147	1.073	1.2	0.834	96
1.073	324	1.156	154	2.029	1.3	0.893	100
1.127	331	1.272	159			0.945	104
1.248	334	1.365	162			0.995	108
1.338	352	1.462	165	Plate #	34	1.033	112
1.431	361	1.533	167			1.092	116
1.538	370	1.610	169	2 <b>a</b>	Nx10 <sup>3</sup>	1.130	120
1.606	375		-	0.345	56	1.172	124
1.675	380			0.385	58	_ , _,	
	•	Plate :	<del> </del>  31	0.394	60		
			^	0.405	66	Plate #	36
Plate	<del>#</del> 29	2 <b>a</b>	Nx103	0.436	74		<u>~</u>
		0.291	1	0.460	80	_2a	Nx10
_2a_	Nx10 <sup>2</sup>	0.338		0.530	88	0.384	51
0.246	<del>6</del>	0.376	2 3	0.580	94	0.422	<u>5</u> -6
0.269	15	0.419	4	0.644	98	0.453	60
0.289	20	0.461	5	0.688	104	0.475	65
0.305	29	0.508	5 6	0.744	108	0.515	70
0.331	35	0.560	7	0.791	114	0.560	75
0.346	40	0.653	<b>?</b> 8	0.857	122	0.583	80
0.386	<i>5</i> 0	0.698	9	0.920	125	0.630	86
0.414	61	0.783	10	0.990	130	0• <b>68</b> 8	92
0.492	75	0.897	11	1.031	135	0.799	100
0.569	84	1.025	12	1.087	136	0.840	104
0.710	96	1.269	13.2	1.133	137	0.893	108
0.810	102	1.360	13.5	1.190	146	0.915	112
0.873	105	1.482	13.8	1.256	150	0.965	116
1.075	113	1.587	14.0	1.288	152	1.056	120
1.144	115	1.734	14.2	1.314	154	1.185	124
1.303	120	2015.		1.421	155	1.239	126
1.370	121			1.461	157	1.314	128
1.459	123	Plate	32	1.479	158	1.426	132
1.520	125			1.517	160	1.54?	135
		_2a	<u>Nx10<sup>3</sup></u>	J1	100	1.601	137
		0.341	0.5			2.002	-)1
Plate	#30	0.444	1.2	Plate #	<b>3</b> 5		
	3	0.490	1.5			Plate #	37
<u> 2a</u>	Nx10	0.587	2.0	2a	Nx10 <sup>3</sup>		
0.258	35	0.727	2.5	0.257	7	_2a	N=10 <sup>3</sup>
0-304	47	0.999	3.0	0.260	20	0.241	35
0.334	56	1.302	3.2	0.280	30	0.317	50
0.359	64	1.583	3.3	0.341	40	0.379	60
0.411	78	1.843	3.35	0.394	48	0.416	70
	•	•		= - 27 +	••	- IAV	, •

Plate	#37 cont.	Plate #		Plate	#42 cont.	Plate	#44 cont.
<u> 2a</u>	Nx103	_2a	Nx10 <sup>3</sup>	2 <b>a</b>	Nx10 <sup>3</sup>	2a_	Nx103
0.477	80	0.266	13	1.178	30	0.663	20.0
0.512	90	0.302	20	1.530	32	0.716	21.0
602	100	0.328	27	1.774	33	0.876	22.0
1.548	105	C•397	34			0.913	22.5
0.703	110	0.452	41			0.971	23.0
0.768	116	0.528	48	Plate	#43	1.005	23.5
0.819	123	0.633	54		3	1•08%	24.0
0.951	131	0.717	60	<u> 2a</u>	Nx10	1.152	24.5
1.046	138	0.789	65	0.390	10	1.214	25.0
1.104	141	0•883	70	0.441	11	1.288	25.5
1.227	146	1.115	76	0.482	12	1.391	26.0
1.329	152	1.172	78	0.521	13	1.508	26.5
1.388	156	1.266	80	0.558	14	1.631	27.0
1.520	161	1.416	83	0•636	15	1.818	27.5
		1.625	86	0•687	16	2.062	28.0
	"-0			0.769	17		
Plate	#38			0-829	18		
•	3	Plate $\#$	41	0.889	19	Plate :	45
<u>2a</u>	<u>Nx10<sup>3</sup></u>		3	0.954	20	_	3
0-299	0	<u> 2a</u>	Nx10 <sup>3</sup>	1.009	21	_2a	Nx10 <sup>3</sup>
0-356	10	0.219	0	1.099	22	0.247	30.0
0.435	16	0•338	35	1.231	23	0.234	30.5
0.487	22	0.360	40	1.353	24	0.307	31.0
0.552	28	0.415	45	1.5 <b>5</b> 0	25	0.322	31.5
0.626	34	0.443	<i>5</i> 0	1.895	26	0.327	33.0
0.729	40	0.470	<b>5</b> 5			0-354	33.5
0.796	पंप	0-577	65			0.362	34.0
0.893	49	0.642	70	Plate :	<b>#</b> 44	0-364	34.5
1.015	54	0. 685	75		2	0.388	35.0
1.118	<i>5</i> 8	0.756	80	_2a_	<u>Nx10<sup>3</sup></u>	0.595	35.5
1.245	62	0.835	85	0.250	5.0	0.405	36.0
1.369	66	0•939	90	0.247	5.5	0.410	36.5
1.595	71	1.053	95	0.252	6.0	0.422	37.0
		1.245	100	0.266	6.5	0.434	38.0
	<b>to</b> = -	1 <b>.49</b> 9	103	0.268	7•0	0.470	39.0
Plate:	<u>#39</u>	1.653	105	0.274	7•5	0.480	39-5
_	3			0•338	12.0	0.493	40.0
2a	Nx10 <sup>3</sup>			0-390	13.0	0.514	40.5
0.367	6.0	Plate #	<u>+2</u>	0.448	13.5	0.527	41.0
0.486	9.0	_	3	0.434	14.0	0.540	41.5
0.605	13.0	2a ]	<u>1x10<sup>3</sup></u>	0.455	14.5	0.549	42.0
0.700	15.5	0.234	3	0.460	15.0	0.572	42.5
0.792	17.5	0.255	6	0-475	15.5	0.600	43.5
0.930	20.0	0.294	9	0-500	16.0	0.620	44.0
1.067	22.5	0.352	12	0.524	16.5	0.660	45.0
1.167	24.0	0.425	15	0.553	17.0	0•690	46.0
1.246	25.0	0.494	18	0-578	17.5	0.725	46.5
1.335	26.0	0.585	21	0.584	18.0	0.751	47.0
1.438	27.2	0.668	24	0.613	18 <b>.</b> 5	0.772	47.5
1.530	28• <i>5</i>	0.936	27	0.628	19.0	0.814	48.0

Plate	#45 cont.	Plate	#46 cont.	Plate	_	Plate	<u>#52</u>
2a	Nx10 <sup>3</sup>	2 <b>a</b>	$N \times 10^3$	2 <b>a</b>	<u>Nx10</u>	<u> 2a </u>	Nx103
0.834	48•5	2.067	16.0	0.242	0.2	0.261	1
0.846	49.0			0.273	0.5	0.291	2
0.883	49.5			0.305	0.8	0.320	2 3 4
0.906	50.0	Plate:	<u>#47</u>	0.325	1.0	0.355	
0.969	50.5		3	0.382	1.5	0-383	5 6
1.007	51.0	<u> 2a</u>	Nx103	0.427	2.0	0.416	6
1.097	52.0	0.222	0.0	0.453	2.2	0-450	7
1.178	52.5	0.300	2.5	0.523	2.6	0.495	8
1.221	53.0	0.321	3.0	0.635	<b>3.</b> 0	0.530	9
1.292	53.5	0.339	<b>3•</b> 5	0.666	3.1	0.576	10
1.379	54.0	0.362	4.0	0.723	3.2	0.622	11
1.498	54.5	0.376	4.5	0.769	<b>3•</b> 3	0.679	12
1.730	55.0	0.403	5•0	0.845	3.4	0.734	13
2.025	55•5	0.434	5.5	0.930	3-5	0 <b>-80</b> 8	14
		0.458	6.0	1.037	3 <b>•</b> 55	0.882	15
	N 1 - 0	0.486	6.5			0-967	16
Plate	<u>#46</u>	0.511	7•0			1.061	17
_	3	0.550	7• <i>5</i>	Plate :	50	1.176	18
<u>2a</u>	Nx10 <sup>3</sup>	0.565	8 <b>.</b> 0		3	1.240	18•5
0.195	0.0	0.644	9•0	<u> 2a </u>	Nx10 <sup>3</sup>	1.312	19
0.216	0.5	0.701	9•5	0.281	0.2	1.390	19.5
0.233	1.0	0.748	10.0	0 <b>. 298</b>	0.3	1.468	20
0.262	1.5	0.802	10•5	0.337	0-4	1.583	20.5
0.277	2.0	0.864	11.0	0.351	0.5	1.707	21
0.305	2.5	0.944	11.6	0.385	0.6		
0.327	3.0	1.013	12.0	0.411	0.7		
0.351	<b>3•</b> 5	1.127	12.5	0.464	0∙8	Plate:	<u>#53</u>
0.368	4.0	1.303	13.0	0-534	0 <b>•9</b>		3
0.395	4.5	1.805	13.6	0.635	1.0	<u> 2a </u>	Nx103
0.428	5.0			0.918	1.1	0.243	1
0.451	5.5		e I. O	2.230	1.15	0.275	1 2 3
0.466	6.0	Plate;	<u>+48</u>			0.309	3
0.490	6.5	•	3		•	0.343	b.
0.525	7.0	2a	Nx10	Plate	<u>51</u>	0.387	5 6
0.552	7• <i>5</i>	0.261	0.50		3	0.423	
0.591	8.0	0.306	1.0	<u> 28</u>	<u>Nx10<sup>3</sup></u>	0.486	7
0•613 0•645	8• <i>5</i>	0.329	1.3	0.273	800	0-540	8
0.688	9.0	0-345	1.5	0.335	830	0.599	9
0.732	9• <i>5</i>	0.380	1.8	0.391	855 975	0.670	10
0.759	10.0 10.5	0.408	2.0	0.443	87 <i>5</i>	0.756	11
0.819	11.0	0•439 0•472	2.2 2.4	0.527	905	0.851	12
0.895	11.5			0.656	930	0.962	13
0.932	12.0	0•518 0•561	2•6 2•8	0.799	950 970	1.028	13.5
1.002	12.5	0.633	3.0	1.030	970 980	1.101	14
1.052	13.0	0.758	3 <b>.</b> 2	1.202	980 985	1.180	14-5
1.133	13.5	0.758	3•4	1.308 1.429	985	1.266	15
1.196	14.0	1.742	3.5	_*	990	1.366	15.5
1.278	14.5	T-1-5	J- J	1.567	995	1.476	16
1.406	15.0					1.551	16.3
1.611	15.5					1.617	16.5
T- 0TT	<b>-</b> J•J					1.728	16.8

# Plate #54

_	2
_2a	Nx103
0.274	0.5
0.319	1.0
0.362	1.5
_	_
0.408	2.0
0.459	2.5
0.515	3.0
0.596	3.5
0.679	4.0
0.794	4.5
0.963	5.0
1.058	5.2
_	
1.203	5•4
1.286	5.5
1.687	5.7

### APPENDIX B

### Crack Growth Data For Combined Loading,

# 2024-T3 Bare Aluminum

Plate	Thickness	Tensile Load	Transverse Load
Number	(in.)	PSI	LB.
la	0.050	10,000	60
2a	0• 050	10,000	80
3a	0 <b>- 050</b>	n	80
4a.	0 <b>• 050</b>	n	100
5 <b>a</b>	0 <b>- 050</b>	n	<b>100</b>
6a	0 <b>• 050</b>	11	125
7a	0 <b>• 050</b>	H .	125
8a.	0.050	W	150
9a.	0.050	<b>n</b>	150
10a	0 <b>• 050</b>	Ħ	175
lla	0.050	11	175
1.2a	0 <b>.050</b>	•	200
13a	0.050	H	200

Plate #la	Plate #3a cont.	Plate #5a cont.	Plate #7a
2a Nr10 <sup>3</sup>	2a Nr10 <sup>3</sup>	2a Nx10 <sup>3</sup>	2a Nx10 <sup>3</sup>
0.306 0.0	0.452 30.0	0.545 26.5	0.306 0.0
0.331 12.7	0.511 36.8	0.619 31.1	0.332 2.3
0.361 30.2	0.560 43.6	0.714 36.0	0.358 5.0
0.404 50.0	0.616 49.0	0.773 39.2	0.400 8.4
0.455 70.1	0.688 54.3	0.846 43.8	0.429 11.5
0.491 81.5	0.754 60.0	0.934 49.1	0.496 15.1
0.573 100.0	0.826 64.9	1.001 52.5	0.548 18.5
0.615 112.2	0.883 68.7	1.077 55.4	0.604 21.4
0.672 130.6	0.953 72.4	1.161 58.6	0.664 24.6
0.749 145.7	1.012 76.5	1.216 60.7	0.748 28.7
0.798 160.0	1.121 81.4	1.299 64.6	0.812 31.4
0.883 179.8	1.158 85.8	1.385 68.5	0.891 34.4
0.933 200.0	1.235 90.2	1.479 71.8	0.969 36.8
	1.280 94.0	1.555 73.8	1.036 39.2
	1.366 99.0	1.612 75.5	1.085 41.6
Plate #2a	1.447 103.0	1.722 79.0	1.148 44.4
3	1.537 106.9	1.835 82.0	1.232 47.4
2a Nx10 <sup>3</sup>	1.612 110.0	1.922 84.8	1.324 51.3
0.324 0	1.704 114.5	•	1.391 56.3
0.348 10.0	1.814 119.0		
0.393 20.0		Plate #6a	
0.432 28.9			Plate #8a
0.491 37.2	Plate #4a	2a Nx10 <sup>3</sup>	
0.546 44.1	_	0.306 0.0	2a N×10 <sup>3</sup>
0.613 51.4	2a Nx103	0.334 2.6	0.298 0.0
0.677 57.0	0.376 0.0	0•347 5•1	0.320 1.9
0.744 62.3	0.452 7.6	0 <b>.403</b> 9.8	0.359 3.9
0.826 67.6	0.535 13.4	0.443 13.3	0.388 6.4
0.897 72.4	0.615 18.6	0.491 16.7	0.439 9.4
0.954 75.4	0.702 24.6	0• <i>5</i> 37 19•3	0.508 12.3
1.031 79.5	0.829 30.1	0.597 22.7	0.566 15.2
1.084 82.3	0.907 33.6	0.635 25.2	0.625 17.8
1.161 85.9	0.984 37.5	0.691 27.5	0.693 20.8
1.221 88.3	1.071 41.0	0.743 29.4	0.751 23.7
1.287 91.6	1.167 45.2	0.821 32.8	0.815 25.9
1.369 94.3	1.247 49.5	0.884 35.7	0.899 29.2
1.434 96.8	1.336 57.6	0.750 38.0	0.971 32.3
1.524 100.0	1.419 58.6	1.013 40.3	1.027 36.2
1.595 102.9	1.497 61.8	1.089 43.0	1.080 40.0
1.717 106.9	1.615 67.1	1.126 44.8	
1.794 110.0	1.685 71.1	1.223 47.8	
1.856 113.3	1.785 74.0	1.309 50.5	Plate #9a
		1.384 52.6	. 1
TO -1- 10-	TO . A . H	1.459 55.0	2a Nx10 <sup>3</sup>
Plate #3a	Plate #5a	1.558 57.3	0.288 0.0
2a. Nx10 <sup>3</sup>	30 15-103	1.661 59.3	0.311 1.8
$\frac{2a}{0.316} \frac{110^3}{0.0}$	2a Nx10	1.739 61.8	0.332 3.4
0.333 10.0	0.360 8.3		0.370 5.9
0.368 17.0	0•360 8•2 0•422 14•5		0.404 8.8
0.405 24.0	0.474 20.1		0.453 10.9
0-70) ATO	OOTIT EUOT		0.507 13.2

Plate #9a cont.	Plate #11a cont.
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0.682 21.3	0.725 22.6
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# Plate #10a

# Plate #11a

	3
28	Nx103
2a 0-283	0.0
0-304	1.6
-	
0.325	3.1
0.364	5.8
0-406	8.5
	_
0.450	10.2
0.489	12.1
0.538	15.3
0.574	17.4
0.641	-
	20.9
0=737	23.4
0.797	27.4
0-889	30.3
0.942	
•	33.3
1-011	34.7
1.069	35.9
1.160	38.8
	40.6
1.233	40.0

# 2a Nx10<sup>3</sup>

1-332	41.6
1.461	43.5

# Plate #12a

2 <b>a</b> _	Nx103
0.310	0.0
0-334	2.5
0.385	4.9
0-437	8.2
0.504	10.9
0-576	13.9
0.627	17.2
0.724	21.2
0.890	24.1
0-987	26.2
1.057	28.8
1.233	30.2
1.592	31.0

# Plate #13a

2a	Nx103
0-304	0.0
0.330	2.3
0.380	4.1
0.428	6.3
0.474	8.2
0.550	11.7
0-600	14.7
0•688	17.5
0.751	20.0
0.803	22.9
0.862	26.1
0.940	29.2
1.008	32.7
1.070	37.8

# Appendix C

# Material Properties

### 2024-13

Thickness (in.)	Yield Strength (at 0.2% - KSI)	Tensile Strength (KSI)	<pre>% Elongation (2" gage length)</pre>
0.050 Clad	39•7	61.7	12.5
0.050 Bare	56•4	75•5	17•4
0.080 Bare	49.9	77•3	18.7
0.100 Bare	61.8	80.3	18.2
0.100 Clad		Not Available	
0.125 Bare	51•4	78•2	19.5
0.125 Clad		Not Available	
0.160 Clad	48.6	69.2	11.6
7075 <b>-1</b> 6			
Thickness	Yield Strength	Tensile Strength	% Elongation
(in.)	(at 0.0% - KSI)	(K2I)	(2" gage length)
0.050 Clad	63•1	77•5	7.8
0.100 Clad	65•6	71.3	2.2
0.120 Bare	75•4	84-2	9•5